#### **ABSTRACT**

This research project sponsored by the Centre for Agricultural Technology(CAT) is seen as assistance to the Agricultural and Beef Industries which are among the major contributing industries to the Australian economy. The project involves a study of the grass seed industry in Central Queensland by conducting a market survey to determine the existing mechanisation in the seed industry and the problems faced by this group of farmers. The main aim of the study was to arrive at a design of an improved and more effective novel grass seed harvester, specifically designed by considering all the characteristics unique to grass seeds and the details which earlier research had indicated would help to increase the quality and the total yield, from the current 40-60% to about 80-95%. Earlier research on an air-assisted brush type of harvester indicated the problem of having to draw the seeds through the fan before This seemed to cause trauma to the seeds which revoked the seed separation. germination capacity. This research work has therefore been focused on designing a seed separator to be located ahead of the fan which could successfully separate the seeds of various shapes and sizes, from the large volume of conveying air required for suction, before entering the fan. A curved duct concentrator and a uniflow cyclone were selected as part of the experimental work and a theoretical approach was developed to understand and define the problems unique to the separation of the grass seeds. The results of these experiments and their comparison with the previous existing separators and harvesters have been discussed in this thesis. The technique for fan design for the harvester and finally the design specifications of a self driven harvester and a tractor mounted harvester were developed by incorporating all the harvester units, i.e the header unit, the separator and the fan.

# DEVELOPMENT OF A NOVEL HARVESTER FOR GRASS SEEDS AND CEREAL CROPS

### LEENA MARY JOHN

Dissertation submitted in partial fulfilment of the requirements for the degree of Master of Engineering

Central Queensland University
Department of Mechanical Engineering
James Goldston Faculty of Engineering
Rockhampton, Australia

31 July, 1995.

## TABLE OF CONTENTS

ABSTRACT i
TITLE PAGE ii
TABLE OF CONTENTS iii
LIST OF FIGURES ix
LIST OF TABLES xiii
ACKNOWLEDGEMENTS xv
DECLARATION
1.0 INTRODUCTION
2.0 MARKET SURVEY REPORT 11
2.1 INTRODUCTION
2.2 PROBLEMS FACED BY PASTURE SEED INDUSTRY 13
2.2.1 Engineering
2.2.2 Agricultural
2.2.3 Marketing
2.2.4 Ancillary services
2.3 MARKET IDENTIFICATION
2.4 ANALYSIS OF THE MARKET SURVEY 21
2.5 CONCLUSION
2.6 ADVANTAGES AND DISADVANTAGES OF THE CQU
HARVESTER 29
2.7 REVIEW OF THE PASTURE SEED INDUSTRY 31
2.8 HARVESTER SIZES BASED ON THE SURVEY REPORT 31

3.0	FAN T	HEORY	AND DESIGN	33
	3.1	INTROL	DUCTION	33
	3.2	VORTE	X FLOW	33
		3.2.1	Free vortex	33
		3.2.2	Forced vortex	34
		3.2.3	Compound vortex	35
	3.3	DEFINIT	TION OF TERMS USED	35
	3.4	FAN LA	WS	37
	3.5	POINTS	TO BE CONSIDERED FOR FAN	38
		3.5.1	System resistance	38
		3.5.2	Operating point	39
		3.5.3	Fan speed	40
		3.5.4	Fan size	41
		3.5.5	Fan control	42
		3.5.6	Fans in series and parallel	44
		3.5.7	Fan selection with respect to noise	46
	3.6	FAN TY	PES AND THEIR CHARACTERISTICS	48
		3.6.1	Centrifugal fans	48
		3.6.2	Axial fans	51
		3.6.3	Propeller fans	53
		3.6.4	Cross flow or mixed flow or tangential flow fans	54
	3.7	FAN SE	ELECTION	56
	3.8	CRITER	RION FOR FAN DESIGN	59

4.0 SEPAI	RATOR	64
4.1	INTRODUCTION	64
4.2	AIR-SOLID SEPARATION	69
4.3	TYPES OF SEPARATORS	70
4.4	PRINCIPLE OF SEPARATION OF PARTICLE	
	SEPARATORS	70
4.5	CLASSIFICATION OF GAS CLEANING EQUIPMENT	71
	4.5.1 Aero-mechanical dry separators	72
	4.5.2 Cyclones or Centrifugal separators	73
	4.5.2.1 Cyclone classification and description	73
	4.5.2.2 Advantages of the cyclone separators	79
4.6	TYPES OF SEPARATORS SELECTED FOR THE	
	EXPERIMENT	79
4.7	SEPARATOR THEORY	82
4.8	INSTRUMENTATION	89
4.9	CURVED TUBE CONCENTRATOR AND UNIFLOW	
	CYCLONE	90
	4.9.1 Curved tube concentrator in combination with a settling	
	chamber	90
	4.9.1.1 Equipment Description	91
	4.9.1.2 Experimental method	94
	4.9.1.3 Experimental development	98
	4.9.1.4 Conclusion for the Curved Duct	
	Concentrator	110

and the state of t

4.9.2 Uniflow cyclone in combination with a settling	
chamber	111
4.9.2.1 Equipment Description	112
4.9.2.2 Experimental method	115
4.9.2.3 Experimental development of Uniflow cyclone	
model 1	120
4.9.2.4 Experimental development of Uniflow cyclone	
model 2	147
4.9.2.5 Conclusion for the Uniflow cyclone	
concentrator	155
5.0 HARVESTER DESIGN-SCALING TECHNIQUES	156
5.1 CONCLUSIONS AND SUGGESTIONS	156
5.1.1 Header unit	156
5.1.2 Air seed separator	159
5.1.3 Fan	160
5.1.4 The settling chamber	160
5.2 CALCULATED SYSTEM PRESSURE	161
5.3 DESIGN OF VACUUM HARVESTER	162
5.3.1 Notations	163
5.3.1.1 Header unit	163
5.3.1.2 Uniflow cyclone unit	163
5.3.1.3 Fan unit	164
5.3.2 Formulas used	166
5.3.2.1 Header unit	166

5.3.2.2 Uniflow cyclone unit	166
5.3.2.3 Fan unit	167
5.3.3 Design specifications	171
5.3.3.1 Tractor mounted vacuum harvester	172
5.3.3.1.1 Header unit	172
5.3.3.1.2 Uniflow cyclone unit	172
5.3.3.1.3 Fan unit	173
5.3.3.2 Self driven vacuum harvester	175
5.3.3.2.1 Header unit	175
5.3.3.2.2 Uniflow cyclone unit	175
5.3.3.2.3 Fan unit	176
5.4 SIZES AND LENGTHS OF DUCTS REQUIRED FOR THE	
VACUUM HARVESTER	180
5.4.1 Tractor mounted vacuum harvester	180
5.4.2 Self driven vacuum harvester	181
5.5 CONCLUSION AND SUGGESTIONS	183
REFERENCES	185
APPENDIX 1 MARKET SURVEY QUESTIONNAIRE	189
APPENDIX 2 SUGGESTED SPECIFICATIONS	190
1.0 SPECIFICATIONS FOR PROTOTYPE (SELF	
DRIVEN)	190
2.0 SPECIFICATIONS FOR PRODUCTION MODEL (SELF	
DRIVEN)	190
APPENDIX 3 BUDGET FIGURES FOR MANUFACTURING COSTS	192

1.0 BU	DGET FIGURES FOR PROTOTYPE	192
2.0 BU	DGET FIGURES FOR HYBRID FROM SECOND HAND	
PA	RTS (SELF DRIVEN MODEL)	193
3.0 BU	DGET FIGURE FOR TRACTOR MOUNTED MODEL	193
APPENDIX 4	PHOTOGRAPHS OF SEED FLOW THROUGH CURVED	
	DUCT CONCENTRATOR	194
APPENDIX 5	PHOTOGRAPHS OF UNIFLOW CYCLONE MODEL 1	196
APPENDIX 6	PHOTOGRAPHS OF UNIFLOW CYCLONE MODEL 2	199

## LIST OF FIGURES

Figure 1.1	Photographs of existing harvesters	3
Figure 3.1	Effect of change in system resistance	40
Figure 3.2	Variation of performance with change of fan speed	40
Figure 3.3	Effect of increase in fan size (speed constant)	41
Figure 3.4	Flow control by speed regulation	43
Figure 3.5	Comparison of Centrifugal and Axial fan noise spectra	47
Figure 3.6	Fan generated noise versus duty	47
Figure 3.7a	Centrifugal fan characteristics	51
Figure 3.7b	Centrifugal fan characteristics	52
Figure 3.8a	Axial fan characteristics	53
Figure 3.8b	Axial fan characteristics	53
Figure 3.9	Propeller fan characteristics	54
Figure 3.10	Cross flow fan characteristics	55
Figure 3.11	One dimensional flow through a centrifugal impeller	61
Figure 4.1	Schematic diagram of reverse flow cyclone with tangential inlet	76
Figure 4.2	Schematic diagram of Uniflow cyclone	77
Figure 4.3	Cut size of standard cyclone	78
Figure 4.4	Figure of different pasture seed head	80
Figure 4.5	Typical particle trajectory in the curved duct concentrator	84
Figure 4.6a	Drag coefficient as a function of Re No. for a flat plate normal	
	to the flow	87
Figure 4.6b	Drag coefficient as a function of Re No for a smooth sphere	87

Figure 4.7	Top elevation of Spiral Duct Concentrator system placed	
	in horizontal plane parallel to the ground	92
Figure 4.8	Side elevation of Curved Concentrator duct system with gauze	99
Figure 4.9	Top elevation of Curved Duct Concentrator system	
	placed in horizontal plane parallel to the ground	102
Figure 4.10	Variation of air velocity across the Concentrator duct	
	width when in horizontal position, with improved	
	tangential concentrated seed outlet and an effective	
	pressure drop across the take off point and the main duct	105
Figure 4.11	Variation of static pressure and total pressure across the	
	Concentrator duct width when in horizontal position	107
Figure 4.12	Uniflow cyclone model 1 and settling chamber with inlet duct	
	reduced to 0.2x0.08 m and underflow air duct fixed inside the	
	settling chamber at an angle to the air outlet duct	112
Figure 4.13	Uniflow cyclone model 2 and settling chamber	113
Figure 4.14a	Uniflow cyclone model 1. Downward air flow velocity profile	
	across the cyclone cylinder and the settling chamber diameter,	
	when underflow air duct is fixed into the air outlet duct	124
Figure 4.14b	Uniflow cyclone model 1. Downward air flow velocity profile	
	across the cyclone cylinder and the settling chamber diameter,	
	when underflow air duct is fixed into the air outlet duct	125
Figure 4.15	Uniflow cyclone model 1. Tangential air flow velocity profile	
	across the cyclone cylinder and the settling chamber diameter,	
	when underflow air duct is fixed into the air outlet duct	126

Figure 4.16	Uniflow cyclone model 1. Air velocity profile across the
	inlet duct width when underflow air duct is fixed into the
	air outlet duct
Figure 4.17	Uniflow cyclone model 1. Air velocity profile across the air
	outlet duct width when the underflow air duct is fixed into
	the air outlet duct
Figure 4.18	Uniflow cyclone model 1. Air velocity profile across the
	underflow air outlet duct width when the underflow air
	duct is fixed into the air outlet duct
Figure 4.19	Uniflow cyclone model 1. Tangential air flow velocity profile
•	across the cyclone cylinder and the settling chamber diameter,
	with the underflow air duct fixed inside the settling chamber. 136
Figure 4.20a	Uniflow cyclone model 1. Tangential air flow velocity
	profile across the cyclone cylinder and the settling chamber,
	with the underflow air duct fixed inside the settling chamber
	and with the inlet duct size reduced. (Fan operated at 1170 rpm)145
Figure 4.20b	Uniflow cyclone model 1. Tangential air flow velocity profile
	across the cyclone cylinder and the settling chamber,
	with the underflow air duct fixed inside the settling chamber
	and with the inlet duct size reduced. (Fan operated at 880 rpm) 146
Figure 4.21	Uniflow cyclone model 2. Downward air flow velocity profile
	across the cyclone cylinder and the settling chamber with
	the underflow air duct fixed inside the settling chamber (Fan
	operated at 880 rpm)

Figure 4.22	Uniflow cyclone model 2. Tangential air flow velocity profile	
	across the cyclone cylinder and the settling chamber with	
	the underflow air duct fixed inside the settling chamber (Fan	
	operated at 880 rpm)	150
Figure 5.1	Seed removing gate	161
Figure 5.2	Velocity triangles	169

## LIST OF TABLES

Table 4.1	Collection efficiency test with W.F.Millet and without flap,	
	gauze and hopper	101
Table 4.2	Collection efficiency test conducted after rig was placed in	
	horizontal plane with White French Millet seeds	104
Table 4.3	Collection efficiency test conducted with Buffel grass seeds .	104
Table 4.4	Collection efficiency test conducted on the Spiral duct	
	Concentrator placed in horizontal plane	109
Table 4.5	Measured and calculated values of volumetric flow rate and air	
	velocity in the inlet, underflow and main air outlet ducts	109
Table 4.6	Collection efficiency test readings on Uniflow cyclone model 1	
	with underflow duct fixed into air outlet duct	121
Table 4.7	Measurement of dynamic and static pressures, inlet velocity,	
	volumetric flow rate at the inlet duct using a pitot-static tube	
	and hot wire anemometer (Inlet duct size of 0.2x0.2 m)	127
Table 4.8	Measurement of dynamic and static pressures, volumetric	
	flow rate, velocity at the main air outlet duct (Outlet duct size	
	of 0.2x0.2 m)	129
Table 4.9	Measurement of volumetric flow rate and air velocity in the	
	circular underflow duct having diameter 0.1 m	129
Table 4.10	Inlet duct readings taken with hot wire anemometer and	
	pitot static tube (Inlet duct size of 0.2x0.2 m)	138
Table 4.11	Outlet duct readings taken with hot wire anemometer and	

	pitot static tube (Outlet duct size of 0.2x0.2 m)	138
Table 4.12	Measurement of volumetric flow rate and air velocity across	
	inlet duct (0.2x 0.8 m) at different fan speeds	142
Table 4.13	Measurement of volumetric flow rate and air velocity across	
	outlet duct (0.2x0.2 m)	143
Table 4.14	Measurement of air velocity across underflow air outlet duct at	
	different fan speeds ( $\phi = 0.1 \text{ m}$ , Area = 0.008 m <sup>2</sup> )	144
Table 4.15	Measurement of volumetric flow rate and air velocity in	
	cyclone inlet duct at 880 rpm	151
Table 4.16	Measurement of volumetric flow rate and air velocity in air	
	outlet duct at 880 rpm	151
Table 4.17	Measurement of volumetric flow rate and air velocity in the	
	underflow air outlet duct at 880 rpm	152
Table 4.18	Concentration test for White French Millet	154

#### **ACKNOWLEDGEMENTS**

I would like to thank the Central Queensland University, the Centre for Agricultural Technology(CAT) and the Queensland Department of Primary Industries(QDPI) for their financial support.

I would like to thank all those who supported and assisted me in the successful completion of this thesis.

Thanks to my supervisors, Mr Dudley Roach, Mr Jim Stevens and Mr John Wildin whose help and assistance with the subject and other matters was invaluable to me and whose timely suggestions helped to complete the project in time.

Thanks to Mr Col Friend who indeed was a good friend to me and assisted me in the lab, in organising and getting things done in time without much delay. Thanks also to Mr Keith Howard and Mr Col Gittins who without any complaint were always ready to listen and assist me in making the changes in the experimental model whenever needed and who at times came up with good suggestions. Thanks to Mr Ray Kerneay and Mr Ken Morrison who helped me out with the experiments at times when no one else was available.

Thanks to Mr Austin Dobson of Kawana Engineering, who assisted me at the start of my project by explaining and showing me the work that had been done on air assisted grass seed harvester by his workshop up until now.

Thanks to Dr Masud Khan for his assistance with literature reviewing.

Thanks to Mr Colin Cole for his friendship and support through the project.

Thanks to my husband Mr Rajan Mathew and my daughter Preethi Mathew, who were patient and of great moral support to me throughout the project.

#### **DECLARATION**

This thesis covers the development and sets a specification for a Novel Vacuum Harvester for pasture seeds. The main area concentrated upon and developed is the air-solid separator unit of this harvester. The development of such a harvester would be of invaluable contribution specifically to the Pasture seed industry and the Beef industry and to the Australian Agricultural industry at large as 50% of its total income is from the pasture seed industry both directly by sale of quality seeds and indirectly by pasture improvement to the beef industry. If the separators designed prove to be only 50% as successful, as indicated in the laboratory experiments, it would help in solving many of the problems presently experienced by farmers in economising the harvester costs, eliminating considerable time lost in post harvest treatments and in stabilising the pasture seed industry.

This new harvester has been designed for a project supported by the Central Queensland University, C.A.T and D.P.I. It has been developed by taking into consideration the various problems faced by the farmers, seed merchants and consumers, to demonstrate the efficiency of this new harvester. Once the efficiency, easy operation and maintenance free parts of this harvester are fully established, it is believed that this harvester will become more popular and widely used in the pasture seed industry.

The work contained in this thesis is a direct result of the experiments carried out by the author and has not been previously submitted for a degree or diploma at any other tertiary institution to the best of my knowledge. This thesis contains no material previously published by another person except where due reference is made.

Leena Mary John

hem My John